

Evolution of Deformation Resistance  $s$

- $s = h_0 \left| 1 - \frac{s}{s^*} \right|^a \text{sign} \left( 1 - \frac{s}{s^*} \right) \dot{\epsilon}^p$
- Describes dynamic hardening and softening of the material.
- $s$  evolves depending on proximity to  $s^*$  and flow activity.

Note: Constants  $A, Q, m, j, h_0, s^*, n, a$  are material-specific and fitted to experimental creep/strain rate data.

Main Equations of Wang's An

#### Flow Rule (Plastic Strain Rate)

- $\dot{\varepsilon}^p = A \exp\left(-\frac{Q}{RT}\right) \left[ \sinh\left(\frac{j\sigma}{s}\right) \right]^{1/m}$
- Plastic strain rate increases with stress and temperature.
- No explicit yield surface; flow occurs at all nonzero stresses.

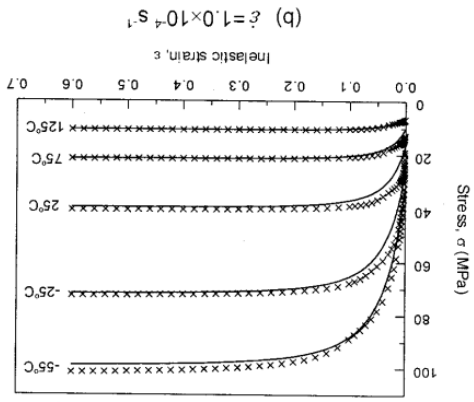
#### Deformation Resistance Saturation $s^*$

- $s^* = \hat{s} \left( \frac{\dot{\varepsilon}^p}{A} \exp\left(\frac{Q}{RT}\right) \right)^n$
- Defines the steady-state value that  $s$  evolves toward.
- Depends on strain rate and temperature.

Key Insights from Wang (2001)

- "At lower strain rates, recovery dominates... the stress levels off early."
- "At high strain rates, hardening dominates, and the stress grows continuously."

Anand's model smoothly captures strain-rate and temperature dependence of solder materials.



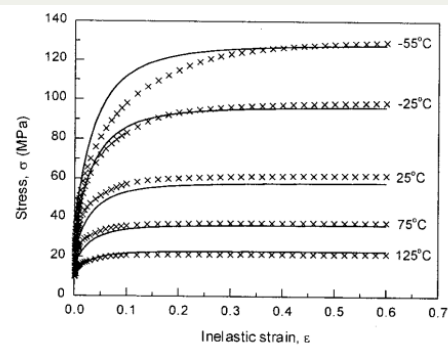
Abstract

This book is a compilation of projects of Michael Raba and can be found at: <https://michaellraba.github.io/talks/>

### Observed Behavior

- **Top Graph (a):**  $\dot{\epsilon} = 10^{-2} \text{ s}^{-1}$
- High strain rate  $\rightarrow$  higher stress
- Recovery negligible  $\rightarrow$  pronounced hardening
- **Bottom Graph (b):**  $\dot{\epsilon} = 10^{-4} \text{ s}^{-1}$
- Lower strain rate  $\rightarrow$  lower stress at same strain
- Recovery and creep effects more significant

**Model Accuracy:** Lines = model prediction, X = experimental data



(a)  $\dot{\epsilon} = 1.0 \times 10^{-2} \text{ s}^{-1}$

Why Wang's Paper Matters

Wang's unified viscoplastic framework to model solder behavior. The model can be reduced and fitted from experiments. The theory is then implemented into engineering-scale implementation. Solder joints in microelectronic packages (chip on PCB, soldered chip on PCB, soldered chip on chip, etc.).

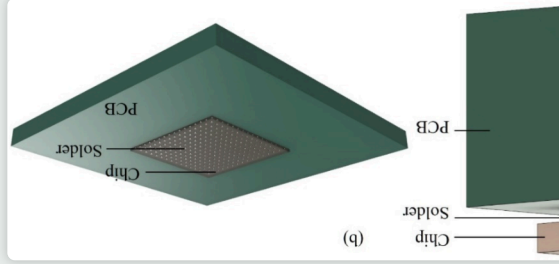


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- Applies An
- Anand's m
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- Targets so  
connection



Source: Wang, C. H. (2001). "A Unified Creep-Plasticity Model for Solder Alloys."  
DOI: 10.1115/1.1371781



Anand Model: Viscoplasticity

Michael Raba, MSc Candida

Created: 2025-0

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## and its Application to Solder Joints

te at University of Kentucky

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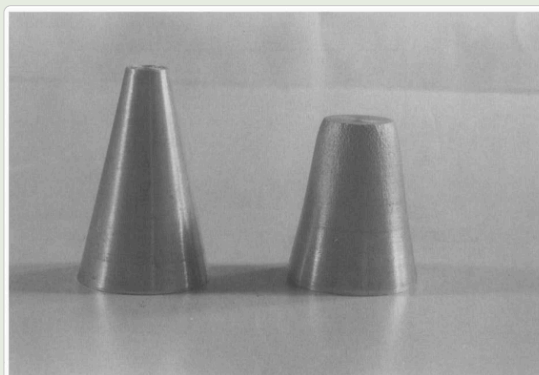
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### **Constitutive Equations for Hot-Working of Metals**

**Author:** Lallit Anand (1985)

**DOI:** 10.1016/0749-6419(85)90004-X

*One of the foundational papers in thermodynamically consistent viscoplasticity modeling—especially significant in the context of metals subjected to large strains and high temperatures.*



**Fig. 25.** 1100 aluminum state gradient specimens before and after testing.

What Happens in the Relaxed Configuration?

- The elastic deformation gradient  $F^e$  is measured from the relaxed state to the current deformed state.
- Elastic strain measures like  $C^e$  and  $E^e$  are defined in this configuration.
- The Kirchhoff stress  $\mathbf{T}$  is naturally associated with the relaxed volume.
- Plastic flow is accounted for separately through the plastic velocity gradient  $\mathbf{L}^p$ .

Summary:

cleanly, enabling proper definition of thermodynamics and

Image Reference

Values are from correspond to 60Sn40Pb solder parameters used in Anand's model:

- $S_0$ : Initial deformation resistance
- $Q/R$ : Activation energy over gas constant
- $A$ : Pre-exponential factor for flow rate
- $\xi$ : Multiplier of stress inside sinh
- $m$ : Strain rate sensitivity of stress
- $h_0$ : Hardening/softening constant
- $g$ : Coefficient for saturation stress
- $n$ : Strain rate sensitivity of saturation
- $a$ : Strain rate sensitivity of hardening or softening

#### Numerical Values

- $S_0 = 5.633 \times 10^7 \text{ Pa}$
- $Q/R = 10830 \text{ K}$
- $A = 1.49 \times 10^7 \text{ s}^{-1}$
- $\xi = 11$
- $m = 0.303$
- $h_0 = 2.6408 \times 10^9 \text{ Pa}$
- $\hat{s} = 8.042 \times 10^7 \text{ Pa}$
- $n = 0.0231$
- $a = 1.34$

These constants match Wang's paper for modeling 60Sn40Pb viscoplasticity.

#### Context for the Relaxed Configuration

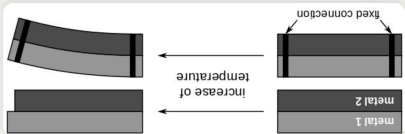
- The relaxed configuration represents the material after removing plastic deformations but before applying new elastic deformations.
- It is introduced to separate permanent plastic effects from recoverable elastic effects.
- All thermodynamic potentials, internal variables, and evolution laws are defined relative to this frame.
- The relaxed state provides a clean, natural reference for measuring elastic strain  $E^e$  and computing dissipation.

#### Sum

- The relaxed configuration isolates elastic responses from plastic evolution laws.

Why Subtract the Thermal Term?

- Thermal expansion creates strain even without external forces.
- Without subtracting  $\epsilon^p$ , the model would falsely attribute thermal strain as mechanical stress.
- Subtracting isolates the true mechanical response from thermal effects.



rain without force.  
mechanical strains generate stresses.  
del physically accurate during heating and cooling.

Forward Euler Explicit time in

Initialization

- Material constants:  $A, Q/R, j, m, h_0, \hat{s}, n, a, E$
- Strain rate:  $\dot{\epsilon}$
- Temperature set:  $\{T_i\}$
- Set:  $\epsilon^p(0) = 0, \quad s(0) = \hat{s}$

Time Evolution Loop

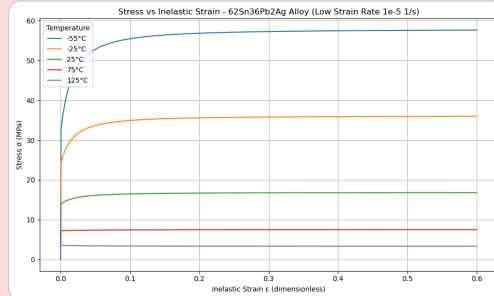
1.  $\epsilon^{\text{total}}(t) = \dot{\epsilon} t$
2.  $\sigma^{\text{trial}} = E(\epsilon^{\text{total}} - \epsilon^p)$
3. Compute  $x = \frac{\sigma}{j\sigma}$
4. Approximate  $\sinh(x)$  (linearize if  $|x| \ll 1$ )
5.  $\epsilon^p = A e^{-Q/RT} (\sinh(x))^{1/m}$

### Plastic Flow & Resistance Evolution

6.  $s^* = \hat{s} \left( \frac{\dot{\varepsilon}^p}{A} e^{Q/RT} \right)^n$
7.  $\dot{s} = h_0 \left| 1 - \frac{s}{s^*} \right|^a \text{sign} \left( 1 - \frac{s}{s^*} \right) \dot{\varepsilon}^p$
8. Update:  $\varepsilon^p(t + \Delta t) = \varepsilon^p(t) + \dot{\varepsilon}^p \Delta t$
9. Update:  $s(t + \Delta t) = s(t) + \dot{s} \Delta t$
10. Record  $(\varepsilon_{\text{total}}, \sigma_{\text{trial}})$

### Termination

- Stop when  $\varepsilon_{\text{total}} \geq \varepsilon_{\text{max}}$
- Plot  $\sigma$  vs  $\varepsilon$  for all  $T_i$



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6 . 1

Stress Evolution ar

### Stress Evolution and Thermal Effects

In the stress evolution equation,

$$\dot{\mathbf{T}} = \mathbb{L} [\mathbf{D} - \mathbf{D}^p] - \mathbb{I} \dot{\theta},$$

the term  $\mathbb{I} \dot{\theta}$  represents the stress change that would occur due to pure thermal expansion alone, without any mechanical loading.

### Summary:

- Thermal expansion induces stress
- Subtracting  $\mathbb{I} \dot{\theta}$  ensures only r
- This keeps the constitutive mo

Material Tensors and Operators

- $\mathbb{L} = 2\mu\mathbb{I} + (\kappa - \frac{2}{3}\mu)\mathbb{1} \otimes \mathbb{1}$  — isotropic elasticity tensor
- $\mathbb{L}\mathbf{D}$  represents how instantaneous strain rates generate stresses according to the elastic material's stiffness properties.
- $\mu = \mu(\theta)$ ,  $\kappa = \kappa(\theta)$  — temperature-dependent moduli
- $\mathbb{H} = (3\alpha\kappa)\mathbb{I}$  — stress-temperature coupling
- $\alpha = \alpha(\theta)$  — thermal expansion coefficient
- $\mathbf{D} = \text{sym}(\nabla\mathbf{v})$  — stretching tensor
- $\mathbf{W} = \text{skew}(\nabla\mathbf{v})$  — spin tensor
- $\mathbf{I}$  = fourth-order identity tensor
- $\mathbf{I}$  = second-order identity tensor

nann derivative to ensure frame indifference.  
ed by isotropic fourth-order tensor  $\mathbb{L}$ .  
duces additional stress through  $\mathbb{H}\dot{\theta}$ .

```
import numpy as np
import matplotlib.pyplot as plt
from scipy.integrate import solve_ivp

# Material constants for 62Sn36Pb2Ag solder alloy
A = 2.24e8 # 1/s
Q_n = 11200 # K
T_m = 13 # dimensionless
n = 0.21 # dimensionless
n0 = 1.02e10 # Pa
s0 = 0.47 # Pa
s_hat = 0.47 # Pa
n = 0.0277 # dimensionless
E = 5.2e10 # Pa (elastic modulus)

# Temperatures in Kelvin
T_c = [-55, -25, 25, 75, 125]
T_list = [T + 273.15 for T in T_c]

# Simulation parameters
strain_rate = 1e-5 # 1/s
eps_total_max = 0.6
T_max = eps_total_max / strain_rate
time_steps = 10000

# Define the ODE system
def system(t, Y):
    eps, s, x = Y
    eps_trial = t * strain_rate + eps
    sigma_trial = E * (eps_trial - eps)
    x = s * sigma_trial / s
    if np.abs(x) < 0.01:
        s_hat = x
    else:
        s_hat = np.sinh(np.clog(x, -30, 30))
    sinh_x = np.max(abs(sinh_x), 1e-10)
    dep_p = A * np.exp(-0.8 / T) * sinh_x**(1/n)
    s_hat = np.exp(1 - s_hat**2 / A * np.exp(0.8 / T))**n
    ds = s_hat * np.exp(1 - s_hat**2 / A * np.exp(0.8 / T))**n
    return [dep_p, ds]

# Plotting
return [dep_p, ds]
```

Forward Euler Scher

me for Anand Model

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6 . 2

Evolution Equati

Stress Evolution Equation (Rate form of Hooke's Law)

$$\overset{\nabla}{\mathbf{T}} = \mathbb{L} [\mathbf{D} - \mathbf{D}^p] - \Pi \dot{\theta}$$

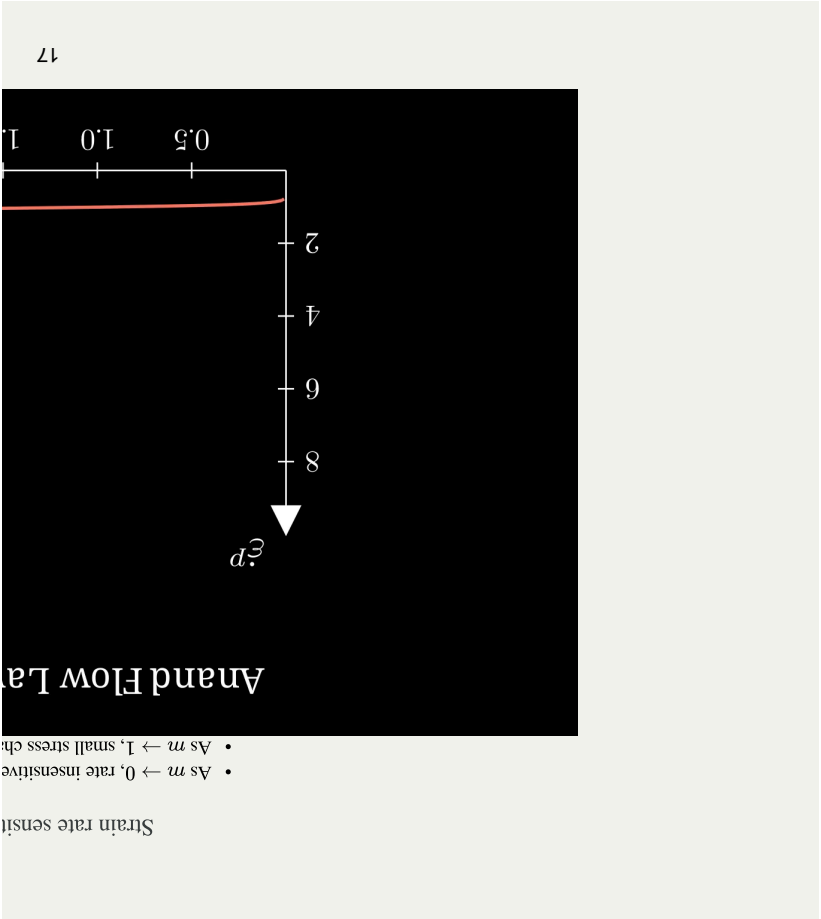
(rate-form Hooke's law for finite deformation plasticity, with frame-indifference enforced through the Jaumann rate.)

Jaumann Rate Definition

$$\overset{\nabla}{\mathbf{T}} = \dot{\mathbf{T}} - \mathbf{W}\mathbf{T} + \mathbf{T}\mathbf{W}$$

**Summary:**

- Stress rate follows Jaum
- Elastic response govern
- Thermal expansion intro



Plastic Strain Rate (magnitude form)

$$\dot{\epsilon}^p = A \exp \left( -\frac{H\theta}{Q} \right) \left[ \sinh \left( \zeta \frac{s}{\sigma} \right) \right]^{1/m} \left( \frac{3}{2} \mathbf{T}' \right)^{1/2} \left( \frac{\sigma}{2} \right)^{1/2}$$

Full Flow Rule with Hyperbolic Sine

$$\mathbf{D}^p = A \exp \left( -\frac{H\theta}{Q} \right) \left[ \sinh \left( \zeta \frac{s}{\sigma} \right) \right]^{1/m} \left( \frac{3}{2} \mathbf{T}' \right)^{1/2} \left( \frac{\sigma}{2} \right)^{1/2}$$

Flow rule

$$\dot{\epsilon}^p = \gamma^p \left( \frac{\mathbf{T}'}{2} \right)^{1/2} \left( \frac{\sigma}{2} \right)^{1/2}, \quad \dot{\epsilon}^p = \left\{ \frac{1}{2} \text{tr}(\mathbf{T}'^2) \right\}^{1/2}$$

Flow rule based on  $\dot{\sigma}/s$ .  
 Stress computed from deviatoric stress.  
 Equivalent stress, but is formally defined without yield point

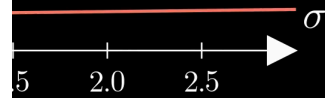
tivity of stress  $m$

:(yield)

ange causes big change in strain rate

ow: Varying  $m$

$$m = 0.05$$



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7.1

Flow

Tensorial Flow Rule (directional form)

$$\mathbf{D}^p = \dot{\epsilon}^p \left( \frac{3}{2} \frac{\mathbf{T}'}{\bar{\sigma}} \right)$$

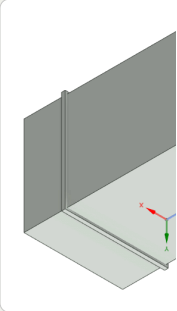
Equivalent Stress Definition

$$\bar{\sigma} = \sqrt{\frac{3}{2} \mathbf{T}' : \mathbf{T}'}$$

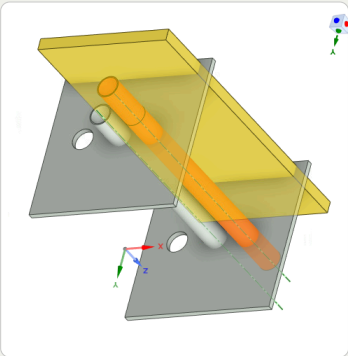
**Summary:**

- Direction given by  $\mathbf{T}'$ .
- Magnitude determined by hyperbolic
- $\bar{\tau}$  represents the effective shear stress
- $\bar{\sigma} = \sqrt{\frac{3}{2} \mathbf{T}' : \mathbf{T}'}$  is the von Mises Eq
- Full flow = **direction**  $\times$  **magnitude**.

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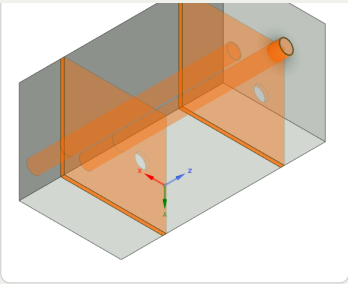


fluid domain



Part 3 — Fiberglass Absorbant (gold)

Part 5 — Final Assembly View



Relaxed Configuration

Kinematics in the Relaxed Configuration

- Elastic deformation gradient:

$$F = F^e F^p \Rightarrow F^e = F F^{p-1}$$

- Elastic right Cauchy-Green tensor:

$$C^e = F^{eT} F^e$$

- Elastic Green-Lagrange strain tensor:

$$E^e = \frac{1}{2}(C^e - I)$$

Summary

- Elastic kinematics and stress measures are formulated in the relaxed configuration
- Stress Power Split allows Anand to cleanly isolate plastic and elastic contributions.
- Green-Lagrange strain tensor  $E^e$  is used because it is defined in the relaxed configuration
- The right Cauchy-Green tensor  $C^e = F^{eT} F^e$  is required to compute  $E^e$  without referencing spatial coordinates

## Stress and Power Quantities

- Kirchhoff stress (weighted Cauchy stress):

$$\tilde{\mathbf{T}} = (\det F) \mathbf{T}$$

- Stress power split:

$$\dot{\omega} = \dot{\omega}^e + \dot{\omega}^p$$

$$\dot{\omega}^e = \tilde{\mathbf{T}} : \dot{\mathbf{E}}^e, \quad \dot{\omega}^p = (C^e \tilde{\mathbf{T}}) : \mathbf{L}^p$$

### Summary:

ed relative to the relaxed configuration, cleanly separating

astic dissipation from elastic storage.

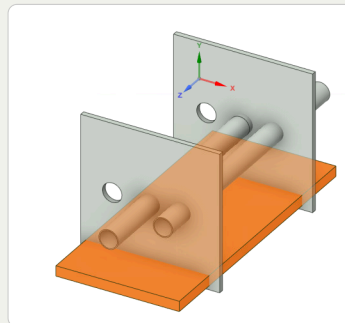
symmetrically captures nonlinear elastic strain relative to the

red as an intermediate to compute  $E^e$  from the elastic  
coordinates

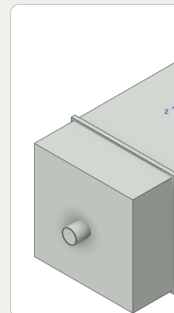
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10 . 2

## Schematic Variants for

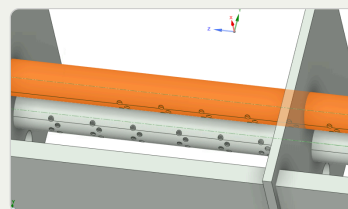


Part 1 — Chamber and Baffle



Part 2 — Flange

Part 4 — Showing perforates (aimed at fiberglass)



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### Key Physical Insights

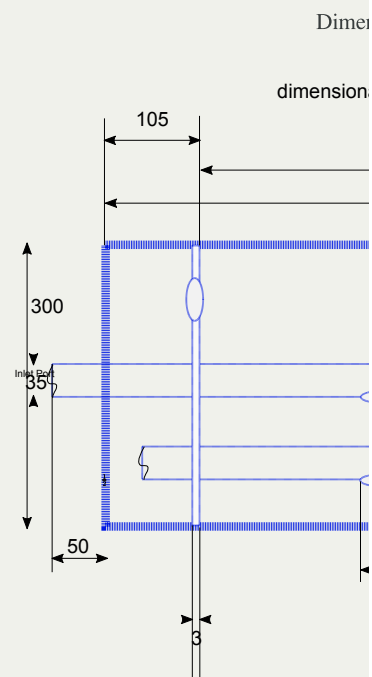
- **Elastic deformations** are recoverable and do not cause entropy production.
- **All dissipation** stems from the plastic flow:  $\dot{\omega}^p$ .
- **Plastic work** increases entropy and governs viscoplastic evolution.

### Summary:

The stress power split ensures that the second law is satisfied by assigning dissipation solely to irreversible processes.

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10 . 3



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- The free energy  $\psi$  is defined relative to the reference configuration.
- State variables like  $E^e, \theta, \underline{g}, \underline{B}, s$  are used as arguments of  $\psi$ .
- Stress is expressed using the second Piola–Kirchhoff tensor  $S$ .
- Dissipation inequality, stress–strain relations, and evolution laws are all written in reference variables.
- Mass density  $\rho_0$  from the reference configuration normalizes all terms.

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- State variables like  $E^e, \theta, \underline{g}, \underline{B}$ ,  $s$  are used as configuration.
- Arguments of  $\psi$ .
- Stress is expressed using the second Piola–Kirchhoff tensor  $S$ .
- Dissipation inequality, stress–strain relations, and evolution laws are all written in reference variables.
- Mass density  $\rho_0$  from the reference configuration normalizes all terms.

- In the reference configuration, all energy storage, stress and with reference-frame quantities for consistency and o

- In the reference configuration, all energy storage, stress and other quantities are defined with reference-frame quantities for consistency and comparison

configuration

### Key Equations in the Reference Frame

- Free energy:

$$\psi = \psi(E^e, \theta, \bar{g}, \bar{\mathbf{B}}, s)$$

- Dissipation inequality:

$$\dot{\psi} + \eta \dot{\theta} - \rho_0^{-1} \mathbf{S} : \dot{\mathbf{E}} + (\rho_0 \theta)^{-1} \mathbf{q}_0 \cdot \mathbf{g}_0 \leq 0$$

- Constitutive relation:

$$\mathbf{S} = \rho_0 \frac{\partial \psi}{\partial E^e}$$

### Summary:

Mass updates, and internal variable evolution are formulated  
objectivity.

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Multicomponent Muff

37

Thermodynamic Quantities

Thermodynamics

- Free energy density:

$$\psi = \epsilon - \theta \eta$$

- Reduced dissipation inequality:

$$\dot{\psi} + \eta \dot{\theta} - \rho^{-1} \mathbf{T} : \dot{\mathbf{T}} + (\rho \theta)^{-1} \mathbf{q} \cdot \mathbf{g} \leq 0$$

- State variables:

$$\{E^e, \theta, \bar{g}, \bar{\mathbf{B}}, s\}$$

with  $E^e$  as elastic strain and  $s$  as internal resistance.

Summary:

- Free energy and dissipation
- Stress power naturally split
- Kirchhoff stress simplifies

Muffler System

at University of Kentucky

5-28 Wed 04:40

### Stress Power and Kirchhoff Stress

- Stress power per relaxed volume:

$$\dot{\omega} = \left( \frac{\rho_0}{\rho} \right) \mathbf{T} : \mathbf{L}$$

- Weighted Cauchy (Kirchhoff) stress:

$$\tilde{\mathbf{T}} = (\det F) \mathbf{T} \quad \text{or} \quad \tilde{\mathbf{T}} = \left( \frac{\rho_0}{\rho} \right) \mathbf{T}$$

- Decomposition of stress power:

$$\dot{\omega} = \dot{\omega}^e + \dot{\omega}^p$$

$$\dot{\omega}^e = \tilde{\mathbf{T}} : \dot{\mathbf{E}}^e, \quad \dot{\omega}^p = (C^e \tilde{\mathbf{T}}) : \mathbf{L}^p$$

govern thermodynamic consistency.  
 s into elastic and plastic parts.  
 stress evolution accounting for volume changes.

Ansys Si  
Simulated Transmission Loss (0-1000 Hz) by a

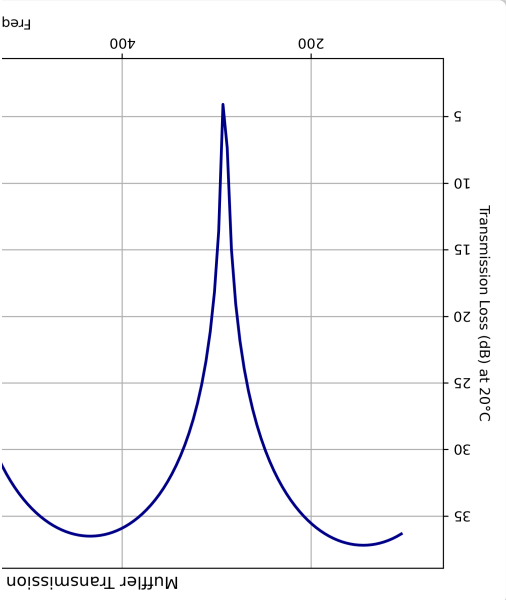
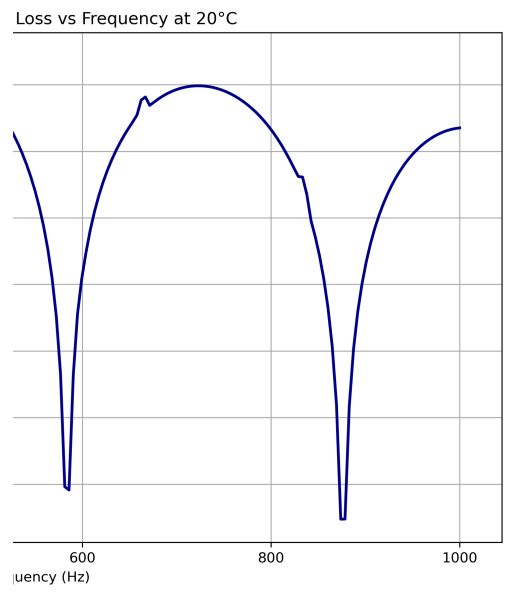


Figure: Transmission Loss curve of the m

mulation

approximating muffler walls as fluid at 20 deg C

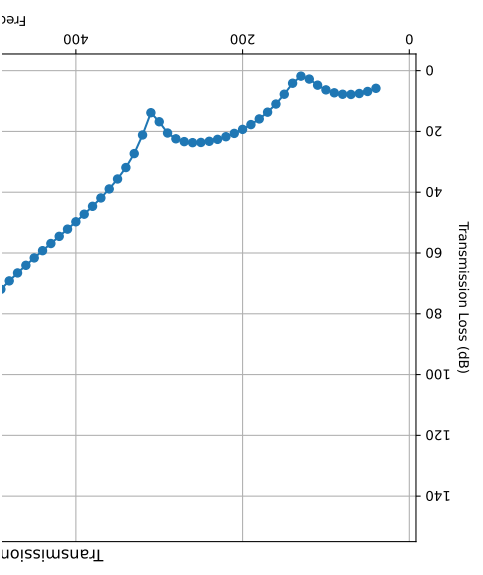


muffler between 5 Hz and 1000 Hz at 20°C.

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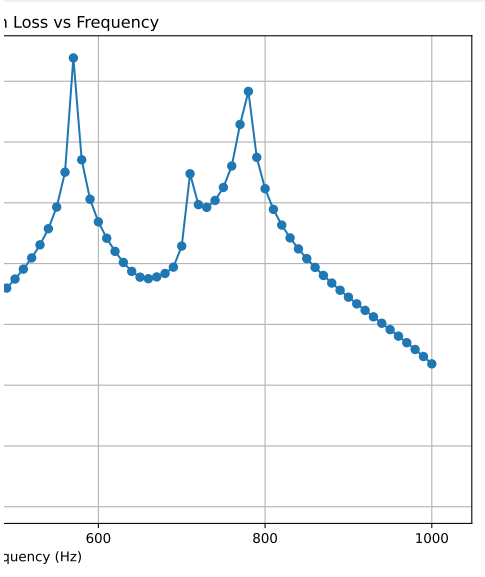
1. Code Execut

Simulab S:  
Simulated Transmission Loss



imulation

s (0–1000 Hz) Simlab model



POD Analysis of T

M. F

Created: 2025-0

Sidlab and Ansys Fi

SIDLAB Model

- File: Mark3Sid.zip
- Created with: SIDLAB 5.1
- [Download SIDLAB File](#)

ences

Works

Wiley; 2014. ISBN: 9781118443125.

ications to Mufflers and Silencers. Cambridge University

1017/9781108840750

nd duct acoustics and were consulted for system modeling.

and transmission loss analysis.

le Download Center

#### ANSYS Simulation

- **File:** Mark-I-MDF-cleaned-data.wbpz
- **Created with:** ANSYS 2023 R2
- [↓ Download ANSYS File](#)

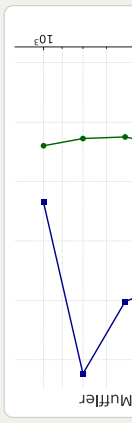
Refer

#### Cited

1. Munjal ML. *Acoustics of Ducts and Mufflers*. 2nd ed. V  
<https://doi.org/10.1002/9781118443125>
2. Dokumacı E. *Duct Acoustics: Fundamentals and Appli*  
Press; 2021. ISBN: 9781108840750. <https://doi.org/10>

**Note:** These references are foundational texts in muffler and  
schematic development, and

Muffled Insertion Loss



Insertion Loss Explanation

Insertion Loss (IL) quantifies how much sound is attenuated when a muffler is added to the system.

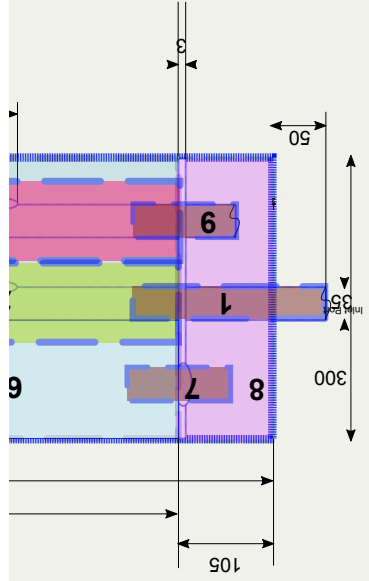
**General formula:**

$$IL = 10 \log_{10} \left( \frac{P_{baseline}}{P_{muffler}} \right)$$

Because our data is already in decibels (dB), this simplifies to:

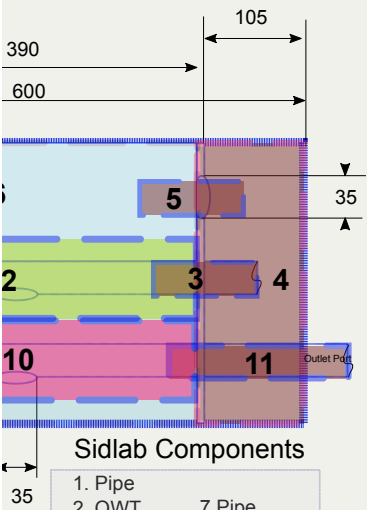
$$IL = Power_{baseline} (dB) - Power_{muffler} (dB)$$

Slab Co dimension



omponents

al units in mm



Sidlab Components

- |         |          |
|---------|----------|
| 1. Pipe | 7. Pipe  |
| 2. QWT  | 8. QWT   |
| 3. Pipe | 9. Pipe  |
| 4. QWT  | 10. QWT  |
| 5. Pipe | 11. Pipe |
| 6. QWT  |          |

Simulated vs Meas

Measured vs Simulated TL



- 1.1.1
1. b7.m

2. initSpectral.m

• reads !

3.  $\hookrightarrow$  initEigs.m

• forms

nstruction is given by

$$\Rightarrow \sum_{m=0}^n \sum_{l=0}^n \alpha_{(n)}(m;l) \Phi_{(n)}(r;m;x)$$

construction can only be recovered by writing for factor  $\gg 0$ .

$$: \gamma) \sum_{m=0}^n \sum_{l=0}^n \alpha_{(n)}(m;l) \Phi_{(n)}(r;m;x)$$

.ayout

in binary files, takes eg m-fft

corrMat, finds eigenvalues

## 2.4. Recon

The reconstruc

$$q(\xi, t) - \bar{q}(\xi) \approx \sum_{j=1}^r a_j(t) \varphi_j(\xi)$$
$$q(r, \theta, t; x) = \bar{q}(r, \theta, t; x) +$$

Since the snapshot pod implementation is not error-free, the re

$$q(r, \theta, t; x) = \bar{q}(r, \theta, t; x) + (\text{factor}$$

1.  $\hookrightarrow$  initPod.m
  - carries out POD calculations (quadrature, multiplication Hellstrom Smitis 2017 for Snapshot POD)
2.  $\hookrightarrow$  timeReconstructFlow.m
  - performs 2d reconstruction + plotSkmr (generates 1d ra

1.2. L<sub>2</sub>

POD Equations

$$\begin{aligned} & \dot{\alpha}^{(n)*}(k; m; t) \, dt \\ & \dot{\alpha}^{(n)*}(k; m; t) \, dt \\ & n; r, t) \mathbf{u}^*(k; m; r, t') \, r \, dr \\ & \dot{\alpha}^{(n)*}(k; m; t) \, dt \end{aligned}$$

ayout 2

ggf between  $\alpha\Phi$ ) according to Papers (Citriniti George 2000 for Classic POD,

dial graph)

2.3. Snapshot F

$$\begin{aligned} &\lim_{\tau \rightarrow \infty} \frac{1}{\tau} \int_0^\tau \mathbf{u}_T(k; m; r, t) \\ &= \Phi_T^{(n)}(k; m; r) \lambda^{(n)}(k; m \\ \mathbf{R}(k; m; t, t') &= \int_r \mathbf{u}(k; r \\ &\lim_{\tau \rightarrow \infty} \frac{1}{\tau} \int_0^\tau \mathbf{u}_T(k; m; r, t) \\ &= \Phi_T^{(n)}(k; m; r) \lambda^{(n)}(k; m \end{aligned}$$

Equations (Fixed)

$$\int_{r'}^{r''} \underbrace{\phi_{*}^{(n)}(r';m;f)}_{\phi_{*}^{(n)}(r';m;f)} \overline{m;f}^{f} \, r'^{1/2} dr'$$

1. obj.CaseId - stores properties like Re, rotation number  $S$ , experiment frequently called vectors (rMat  $r = 1, \dots, 1, 0.5$ )
2. obj.pod - eigen data, used for calculating POD
3. obj.solution - computed POD modes
4. obj.plt - plot configuration
- pipe = Pipe(): creates a Pipe Class. As the functio
- 1.3. Importe

int Switches

ns (above) are called, data is stored in sub-structs:

tal flags such as quadrature (simpson/trapezoidal), number of gridpoints,

2.2. Classic POD

$$\begin{aligned} &\int_{r'} \underbrace{r^{1/2} S_{i,j}(r,r';m;f)}_{W_{i,j}(r,r';m;f)} r'^1 \\ &= \underbrace{\lambda^{(n)}(m,f)}_{\lambda^{(n)}(m;f)} \underbrace{r^{1/2} \phi_i^{(n)}(r;f)}_{\phi_i^{(n)}(r,m;f)} \\ \alpha_n(m;t) &= \int_r \mathbf{u}(m;r,t) r \end{aligned}$$

OD Equations

re used in the above code:

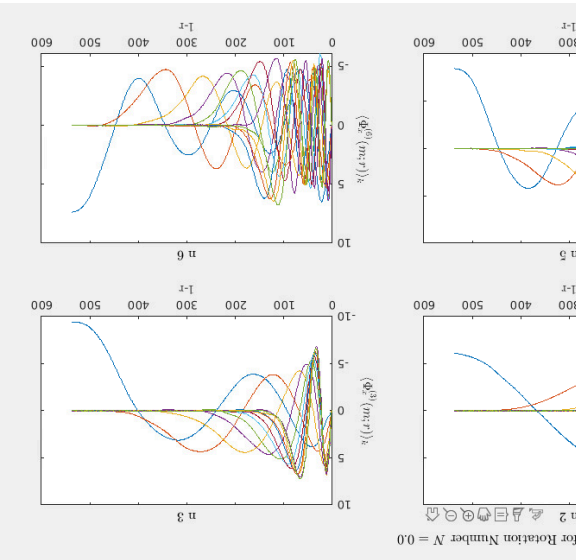
$$r' \mathrm{d}r' = \chi_{(u)}(k; m) \Phi_{(u)}(k; m; r)$$
$$; m; r, t) \mathbf{u}^*(k; m; r', t) \, \mathrm{d}t$$
$$\mathfrak{p}_{(u)}^*(k; m; r) \, r \, \mathrm{d}r$$

in Code Procedure

2.1. Classic Po

The following equations a

$$\int_{r'} \mathbf{S}(k; m; r, r') \Phi^{(n)}(k; m; r')$$
$$\mathbf{S}(k; m; r, r') = \lim_{\tau \rightarrow \infty} \frac{1}{\tau} \int_0^\tau \mathbf{u}(k$$
$$\alpha^{(n)}(k; m; t) = \int_r \mathbf{u}(k; m; r, t) \mathfrak{d}$$

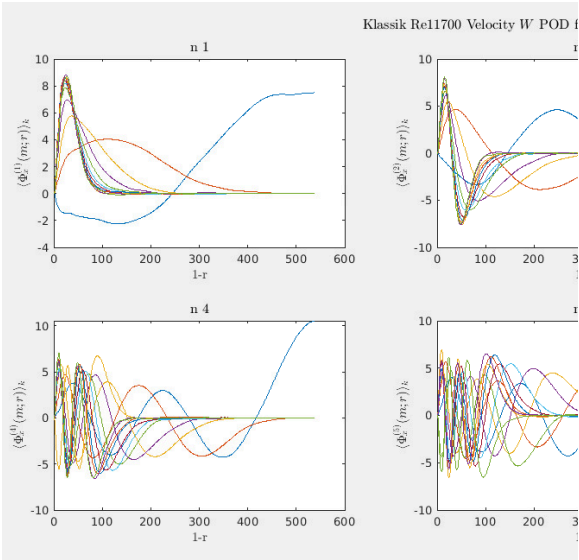


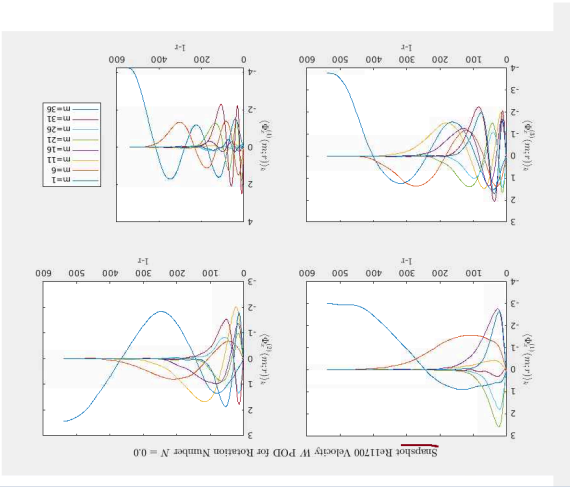
2.5. Reco  
In order to reconstruct in code, caseId.fluctuation = 'off', T]

nstruction

his is incorrect. The necessary use of (factor  $\gamma$ ) is incorrect

4.3. Klassik





3. Der.

To derive the questioned eq

$$\frac{1}{\tau} \int_{\tau}^{\tau} \mathbf{u}_{\tau}(k; m; r, \tau)$$

Substitute  $\mathbf{u}_{\tau}$  w

$$\frac{1}{\tau} \int_{\tau}^{\tau} \Phi_{(l)}^{\tau}(k; m; r) \alpha_{(l)}$$

ivation

uation, consider the integral:

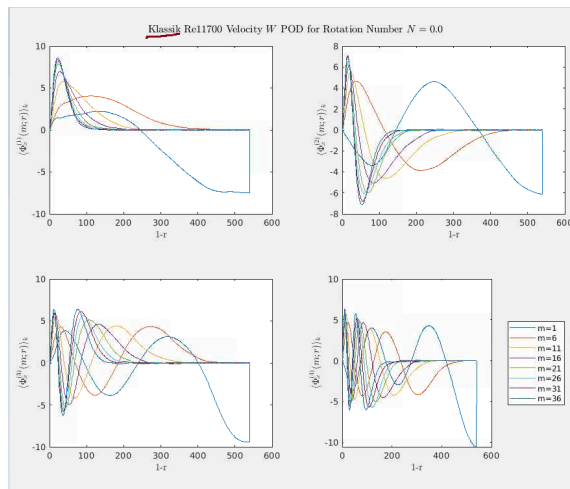
$$t) \alpha^{(n)*}(k; m; t) dt.$$

ith its expansion:

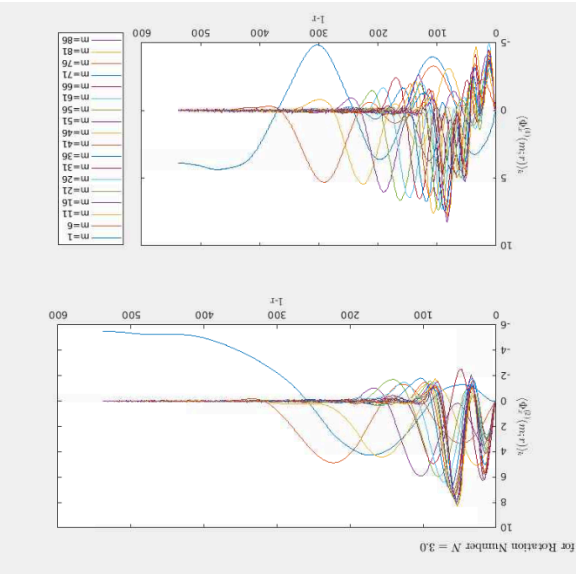
$$^j(k; m; t) \bigg) \alpha^{(n)*}(k; m; t) dt.$$

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#### 4.2. Snapshot-Cl



87



3.1.1. 4 D

Exchange the order of summation and

$$\sum^l_{l=0}\Phi^{\text{T}}_{(l)}(k;m;x)\left(\frac{1}{\tau}\int_0^{\tau}\alpha_{(l)}\right)$$

Due to the orthogonality, namely t

$$\langle a_{(n)}|\alpha_{(p)}\rangle$$

all terms where  $l \neq n$  will vanish, an

$$\Phi^{\text{T}}_{(n)}(k;m;x)\left(\frac{1}{\tau}\int_0^{\tau}\alpha_{(n)}\right)$$

This derivation assumes the normalization of modes and their orthogonality, form that reveals the spatial structure (  $\Phi^{\text{T}}_{(n)}$  )

erivation

1 integration, and apply orthogonality,

$$l)(k; m; t) \alpha^{(n)*}(k; m; t) dt \Big) .$$

that  $\alpha^{(n)}$  and  $\alpha^{(p)}$  are uncorrelated

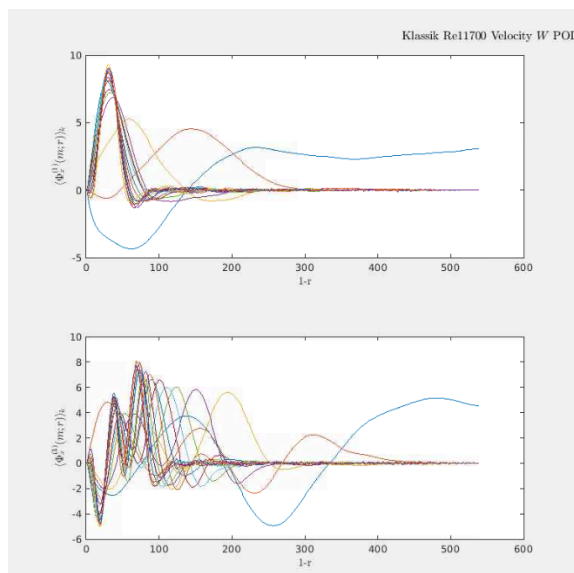
$$= \lambda^{(n)} \delta_{np}$$

and there remains only the  $l = n$  term,

$$(k; m; t) \alpha^{(n)*}(k; m; t) dt \Big) .$$

along with the eigenvalue relationship to simplify the original integral into a sum of each mode scaled by its significance  $(\lambda^{(n)})$ .

#### 4.1. Radi



3.2. 6 D

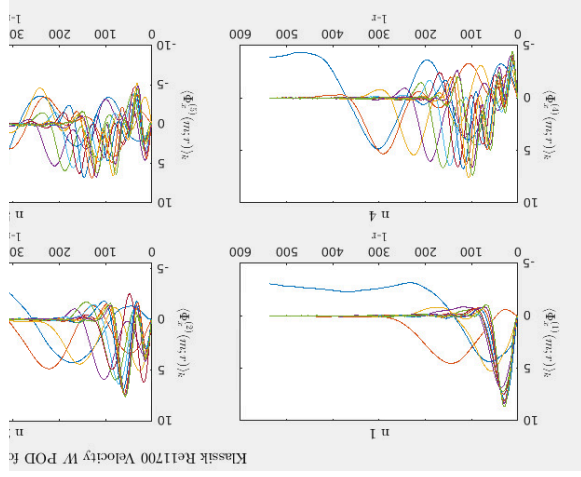
The cross-correlation tensor  $\mathbf{R}$  is defined as  $\mathbf{R}(k; m, t, t') = \int_P \mathbf{u}(k; m, r, [t \times t']$  tensor. The  $n$  POD

$$\lim_{t \rightarrow \infty} \frac{1}{t} \int_0^t \mathbf{u}_r(k; m, r, t) \alpha^{(n)}_r(k; r$$

erivation

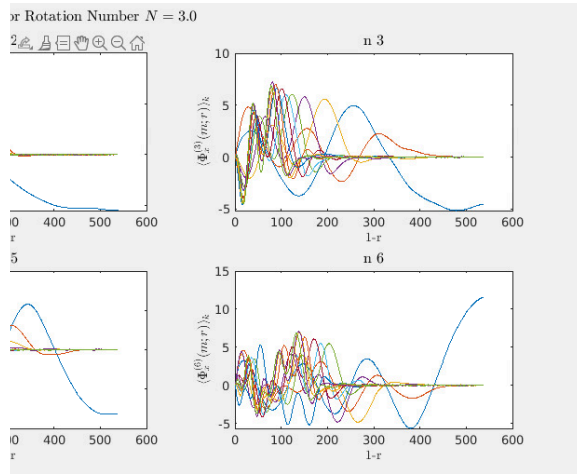
$t)\mathbf{u}^*(k;m;r,t')\,r\,\mathrm{d}r$ . This tensor is now transformed from  $[3r\times 3r']$  to a modes are then constructed as,

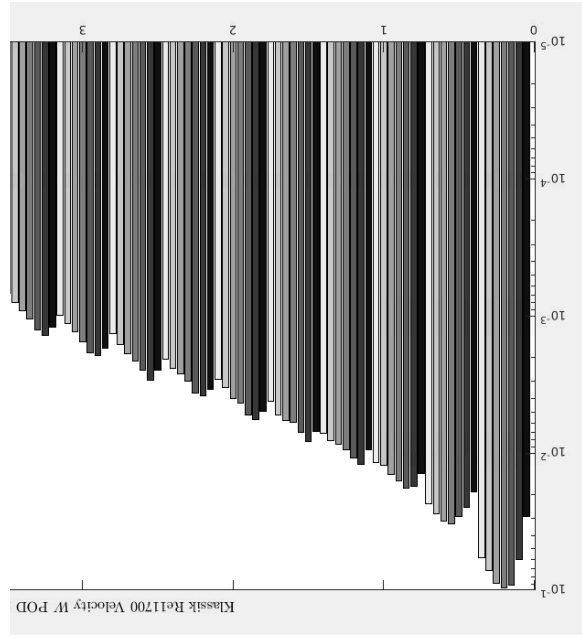
$$n;t)\mathrm{d}t=\Phi_{\mathrm{T}}^{(n)}(k;m;r)\lambda^{(n)}(k;m).$$



4.4. Klassik

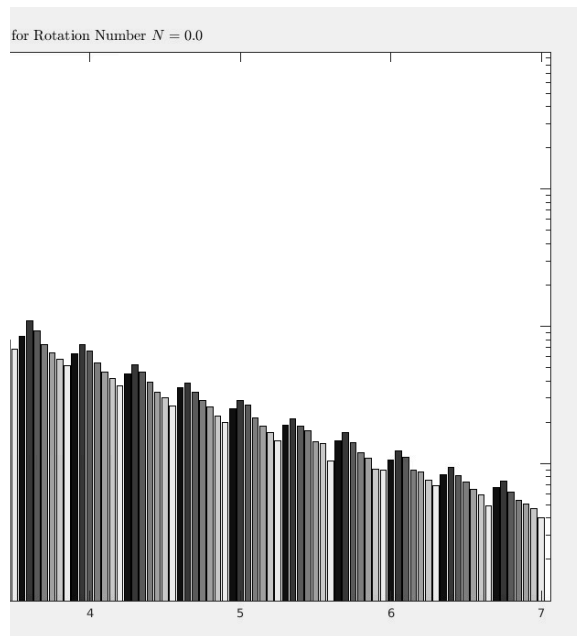
POD S=3.0

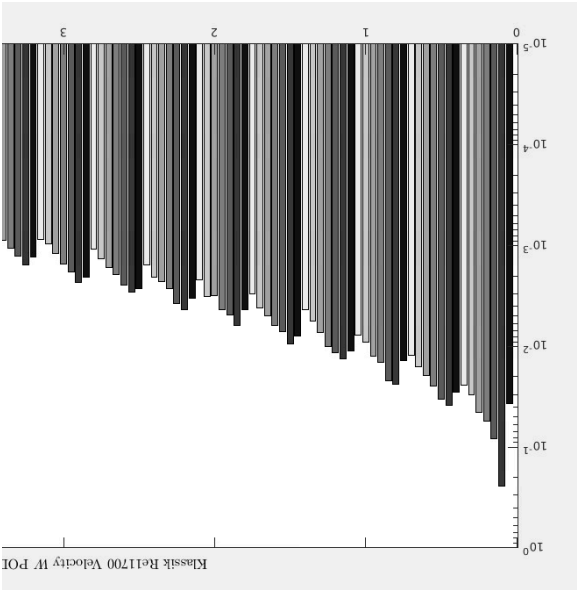




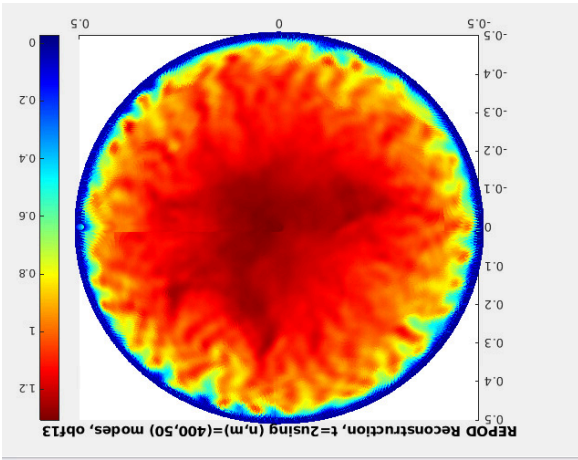
5. Energy r

$\nu=0$  Classic



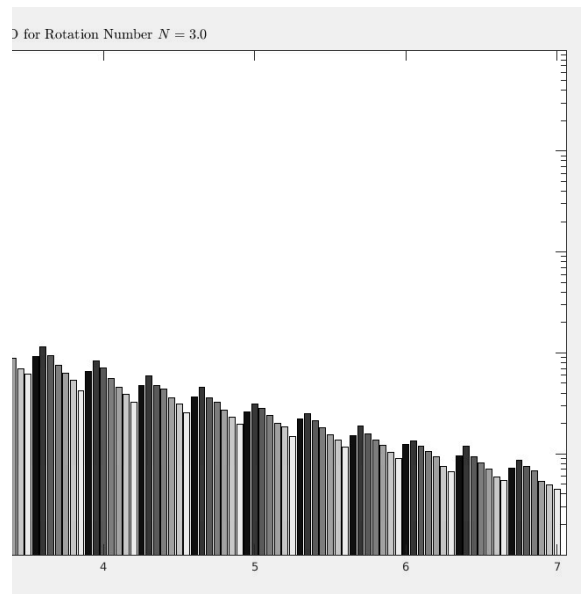


5.1.  $n=3$



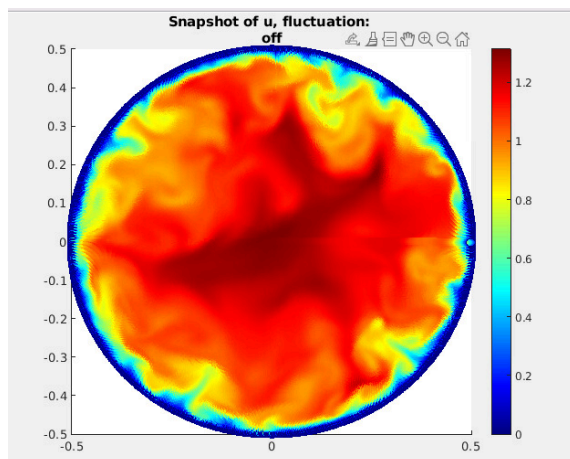
reconstruction

Classic



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6.1. Reco



101



nalysis

98

6. Recon

99